

Temperature measurement and heat load estimation in fusion experiments

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- Why heat load measurements?
- Thermography @ ASDEX Upgrade
- Thermography basics
- Heat load calculation
- Summary



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Why is heat load measurement and control such a serious topic?

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- Heat load exhaust as general problem in fusion devices
- 1990th change from limiter to divertor configurations.
- Disentangling plasma core (performance) and heat exhaust



Larger devices – higher divertor heat load



Heating power increases with about

 $P_{Heat} \sim R^2$

Plasma stored energy increases stronger than with R³

$$W_{mhd} \sim \beta_t B_t^2 R^3$$

Surface to be loaded with radiation:

$$A_{Vessel} \sim R^2$$

Wetted area of the divertor:

$$A_{Div} \sim R \times \Delta x^{\gamma}$$



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- All configurations are in principle technically feasible (forces, shielding)
- The cost-to-benefit ratio has not been assessed so far!

Zohm 2015, Adapted from H. Reimerdes et al., 1st IAEA TM on divertor concepts, 2015

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- Target heat loads are no concern for presently running fusion experiments.
- But are a serious concern for future larger fusion reactors.
 - Higher heat load (MW/m²)
 - Material degradation due to Neutrons
 - Smaller temperature operation range for actively cooled targets



Surface temperature limitations



- Sputtering and erosion during 'normal' plasma wall interaction:
 - Impurity concentration
 - Life time
- Catastrophic increase of released material above a critical temperature:
 - Strong life time reduction
 - Damages
 - The threshold temperature depends on the material.

Establish plasma operation resulting in PFC temperatures below the threshold.

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Interface limitations



Actively cooled targets





- Stationary case: 10 MW/m²
 - The sensitive component is inside the target ...
- Safety issue:
 - The surface temperature is measured.
 - Correlation to the temperature inside the bulk.
 - Thermal model
- Situation might be opposite for pulse like events (ELMs, Disruptions)
 - Surface temperature limits will be exceeded.

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Tolerable heat load?



Relation between material and (plasma) heat deposition



steady state:

$$\Delta T_s = (T_s - T_{cool}) = \frac{q_s}{\kappa} d$$

actuator: profile shape

transient (short vs. transition time (a few seconds)):

$$\Delta T_t = (T_t - T_s) = \frac{2}{\sqrt{\pi}} \frac{q_t \sqrt{\Delta t}}{\sqrt{\kappa \rho c}}$$

actuator: profile shape temporal evolution •minimum thickness •life time •transients

$$q_s^{\max} \le 15MW / m^2$$
$$P = 2\pi R \int q(x) dx$$



See the talk by Aleksey Arakcheev



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Thermography at ASDEX Upgrade

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Physics investigation



Examples





Bremsstrahlung/Reflections





Physics investigation - ELMs



- 1992 Type-I ELMs measured with a fast IR line camera
 - 256 pixel
 - LN₂ cooled
- Since then thermography is a standard diagnostic at AUG



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- How to measure surface temperatures?
- How to optimize the measurement according the task?
 - Physics
 - Machine protection
 - Temperature range and sensitivity

ASDEX Upgrade Max-Planck-Institut für Plasmaphysik Planck's law – correlation between photon flux (power flux) and temperature



$$M_{e}(T,\lambda) = \frac{c_{1}}{\lambda^{5}} \frac{1}{\exp(\frac{c_{2}}{\lambda T}) - 1} \qquad [M_{e}] = \frac{W}{m^{2}\mu m}$$

$$c_1 = 2\pi hc^2 = 3,741 \times 10^{-16} Wm^2$$
 $c_2 = \frac{ch}{k} = 1.438 \times 10^4 \,\mu mK$



- Planck's law
 - Strong non-linear
 - Unique relation between radiation/photon emission of a body and temperature.
 - Depends on the wave length (broad band radiation).
- Select an optimum wavelength:
 - Temperature range.
 - Environment (vacuum, air).
 - Available detectors (costs).

Wien's displacement law Relation between λ_{max} and T

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Planck's formulae - approximated





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Temperature error increases with wavelength and temperature



The measured signal consists of the temperature information and a background signal (fixed and small wavelength window)

$$S([digits]) = K\varepsilon\tau \frac{1}{\exp(\frac{c_2}{\lambda T}) - 1} + S_{Bck}(T_{Bck})$$

temperature of the object:

$$T = \frac{c_2}{\lambda} \frac{1}{\ln(\frac{K\varepsilon\tau}{S - S_{Bck}} + 1)}$$

measuring error ?

$$dT = \frac{\partial T}{\partial K} dK + \frac{\partial T}{\partial \varepsilon} d\varepsilon + \frac{\partial T}{\partial \tau} d\tau + \frac{\partial T}{\partial S_{Bck}} dS_{Bck}$$

$$\frac{\Delta T}{T} = \frac{\lambda T}{c_2} \frac{\exp(\frac{c_2}{\lambda T}) - 1}{\exp(\frac{c_2}{\lambda T})} \left(\frac{\Delta K}{K} + \frac{\Delta \varepsilon}{\varepsilon} + \frac{\Delta \tau}{\tau} + \frac{S_{Bck}}{\frac{K\varepsilon\tau}{\exp(\frac{c_2}{\lambda T}) - 1}} \frac{\Delta S_{Bck}}{S_{Bck}}\right)$$
<1 \approx 1
<1 \approx 1
<> 1 for
<_2 = 1.438 \times 10^4 \mumk

- K calibration factor
- $\boldsymbol{\epsilon}$ emissivity of the object
- $\tau\,$ transmission of the optical system
- S_{Bck} background radiation (the background temperature(s) is constant in time scale of the signal change)



Summary – Wavelength detection



- Typical wavelengths regions for T measurement:
 - Vis/near infrared (vis/NIR, ~ 1 μm)
 - Mid wave infrared (MWIR, ~ 5 µm)
 - Long wave infrared (LWIR, ~ 10 μ m)
- MWIR and LWIR cover temperature range from 500 to 3500 K.
- Vis/NIR covers a 'small' T-range.
- T measurement error:



- Strong error mitigation in the vis/NIR wavelength region:
 - Comparator like behaviour.
 - robustness against change of system parameters (emissivity).



Properly designed thermography system

- Adapted to the measuring requirements
- Calibrated

Are a prerequisite for heat load calculation.

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Heat load calculation

Power flux

- $\vec{q} = -\kappa \operatorname{grad} T \qquad ; \left\lfloor \frac{W}{m^2} \right\rfloor$
- κ heat conduction coefficient

- No direct power measurement.
- Power is calculated from a temperature change

Temperature response – heat conduction equation

$$\rho c_{p} \frac{\partial T}{\partial t} = div(\kappa \operatorname{grad} T) + S$$

$$\rho c_{p} - heat \operatorname{capacity}$$

$$a = \frac{\kappa}{\rho c_{p}} - heat \operatorname{diffusivity}$$

Time constant (Fourier number)

$$F_0 = \frac{at}{d^2}$$
 $F_0 < 1/\pi - \text{semi infinit}$

Heat conduction equation.

- Relation between temperature change and heat flux.
- Solve for appropriate thermal model.

 $\tau = \frac{d^2}{\pi a}$ Time to penetrate into depth d $\tau_{CFC} = \frac{d^2}{\pi a} = \frac{(2cm)^2}{\pi 2cm^2/s} \approx 0.7s$

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From photon flux to heat flux





$$\Delta T_s(t) = \frac{2}{\sqrt{\pi}} \frac{1}{\sqrt{\kappa \rho c}} q_s \sqrt{\Delta t}$$

$$T_s - T_{back} = \frac{d}{\kappa} q_s$$

Stationary Actively cooled target Depends on: Heat conductivity Stationary Thin layer with bad thermal contact Depends on: heat capacity

 $\Delta T_{s} = \rho c d q_{s} \Delta t$



Numerical - heat load calculation

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2D heat flux equation (THEODOR)

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Numerical solved (FTCS, explicit)



- Thermal model with heat transfer edge condition on top and bottom.
- Bottom: clamped to the cooling structure
- Top: to consider surface effects (transients)

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Surface effects



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- Same T(t) as input
- ... but different thermal models
- One order of magnitude difference in the heat flux (transients).

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 No difference in the calculation of the accumulated energy at the target!

- Thermal model and surface conditions
 - Micro structures as pseudo layers
 - Layers
- Wavelength selection

ASDEX Upgrade Max-Planck-Institut für Plasmaphysik Surface effects – micro thermography () • Overestimation of the bulk temperature due to:

Surface morphology





ToreSupra limiter



Laser beam heating, 5 MW/m²

- During surface heating micro spots (a few 10µm) are detected at the surface.
- Detected for plasma exposed and unexposed materials.
- Pattern is constant over at least a few hundred load cycles.
- Layers in deposition areas are hotter compared to the bulk material.
- For the same heat flux!

ASDEX Upgrade Max-Planck-Institut für Plasmaphysik Hot spots result in an artificial temperature increase



R_T – temperature ratio hot spot/bulk R_a – area ratio hot_spot/total area



• The microscopic temperature patterns are fixed over many heating cycles.



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Layer effects



'wrong surface properties':

- Temperature jump results in a strong heat pulse (positive and negative).
- The integral signal gives the deposited energy.
- Correction of top layer effects on time scales longer then the pulse length (energy conservation)

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Thermal model



Bulk material: thermal data known

Numerical:

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$$d = 50 \,\mu m \qquad \kappa_b = 100 \frac{W}{m K} \qquad \kappa_l = 1/10 \,\kappa_b$$

The additional ΔT is 45 K/MWm⁻²

The time constant for such a thin region is short. $\tau = \frac{d^2}{\pi a} \approx 50 \,\mu s$

After this time the time behaviour of the surface temperature follows the heating of the bulk.

Temperature gradient in top of the bulk

$$\Delta T_{layer} = T_s - T_b = \frac{d_l}{\kappa_l} q_s$$

1. $\kappa_l = \kappa_b$

(Nearly) no effect on the measured surface temperature

2.
$$\kappa_l < \kappa_b$$

The surface temperature is increased The derived heat flux is too large if the surface effect is not considered.









Temperature equivalent for Bremsstrahlung. A constant pressure of $n_e T_e = 1 \times 10^{22} \text{ eVm}^{-3}$ is assumed.

$$\frac{dW_B}{d\lambda \ dV} = const \frac{1}{\lambda^2} \frac{Z_{eff} n_e^2}{(T_e)^{1/2}} G_{ff} Exp\left[-\frac{hc}{\lambda T_e}\right]$$

- Bremsstrahlung is emitted from a dense plasma in the SOL and divertor region.
- Strong decrease of Bremsstrahlung contribution between 1 and 5 µm
- Optimum wavelength 5 µm
- Cold and dense plasmas contribute to Bremsstrahlung.
- Reduced target load due to divertor detachment.

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 Heat dissipation is a serious task for the design of a fusion reactor and is a main topic in present research activities.

Summary

- Thermography is a powerful tool for safety and physics investigations.
- Compared to infrared systems, video cameras have a smaller measurement range but are 5 -10 times more robust against changes of the optical system including emissivity and transmission.
- Heat flux = Temperature + thermal model
 - Temperature measurements needs careful interpretation
 - Transients are dominated by surface effects
 - Long lasting events are dominated by the bulk (robust energy conservation)
- The non linearity of Planck's law pronounces hot spots at low temperatures and low wavelengths).
- Heat load calculation is energy conserving
 - As long as the thermal model for the bulk is correct, 'wrong' transient effects will